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Dariusz Horla, Mahmoud Hamandi, Wojciech Giernacki, Antonio Franchi. Optimal Tuning of the Lateral-Dynamics Parameters for Aerial Vehicles with Bounded Lateral Force. *IEEE Robotics and Automation Letters*, 2021, 6 (2), pp.3949 - 3955. 10.1109/LRA.2021.3067229 . hal-02970615v2

**HAL Id: hal-02970615**

**<https://univ-smb.hal.science/hal-02970615v2>**

Submitted on 2 Feb 2021

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# Optimal Tuning of the Lateral-Dynamics Parameters for Aerial Vehicles with Bounded Lateral Force

Dariusz Horla<sup>1\*</sup>, Mahmoud Hamandi<sup>2\*</sup>, Wojciech Giernacki<sup>1</sup>, Antonio Franchi<sup>2,3</sup>

**Abstract**—This letter shows for the first time why it is important and how to optimize the gains of a position controller on board of a fully-actuated aerial vehicle with bounded lateral force, via an auto-tuning approach. In such vehicles, most of the control authority is expressed along a principal thrust direction, while along lateral directions smaller forces can be exploited to achieve full-pose tracking. The nonlinear and hard to model interplay between the constraint imposed on the lateral force and the gains of the position controller is overcome employing the OPTIM-tune calibration method. Several experimental tests, performed fully autonomously during flight, clearly show the practicability and benefits of the approach.

**Index Terms**—Aerial Systems: Mechanics and Control, Aerial Systems: Applications, Motion Control.

## I. INTRODUCTION

UNMANNED Aerial Vehicles (UAV)s have been widely studied in the literature over the past few years, with applications varying between search and rescue [1], fire fighting [2] and, more recently, aerial physical interaction [3].

Most UAV applications rely on the use of *collinear/coplanar* platforms [1], [2], [4], such as quadrotors, hexarotors or octorotors, where all propellers are coplanar and provide thrust in a direction parallel to the platform’s vertical axis. While the use of these platforms allowed the advancement of UAV research in the above mentioned fields, these platforms lack the ability of applying lateral forces and thus need to tilt to move laterally.

Different designs from the literature overcome this limitation by adding additional propellers that can apply lateral forces. Romero et al. [5] add four propellers to a quadrotor along the major lateral directions to move sideways. Similarly, Albers et al. [6] add an extra propeller that produces thrust perpendicular to the four main propellers. Conversely, Ryll et al. [7] apply lateral motion by tilting each of the propellers of a quadrotor independently.

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This work was partially funded by the European Robotics Research Infrastructure Network H2020 INFRAIA TERRINet (2017-2020) under Grant Agreement no. 730994 during translational infrastructure access in TERRINet; by Poznan University of Technology under grant no. 214/SBAD/0220; and by the European Union’s Horizon 2020 research and innovation programme under grant agreement ID: 871479 AERIAL-CORE.

Recently, all the platforms that can apply lateral forces in body frame have been grouped in the abstract class of Bounded Lateral Force (BLF) UAVs, firstly introduced in [8]. Unlike the more popular quadrotors, these platforms can apply a lateral force in their body frame thanks to tilted propellers, similar to the example platform shown in Fig. 1. Since BLF platforms can move laterally without the need of tilting, they can tilt without the need of moving laterally, and can interact with the environment with multiple force directions while maintaining an independent desired orientation. In its simplest and more effective representation, among the ones introduced in [8], a BLF platform’s feasible force set is modeled as a cylinder which radius, summarized by the parameter  $\overline{f_{xy}}$ , which represents the maximum allowable lateral force in any horizontal direction.

The geometric controller presented in [8] requires the inertial parameters of the platform, and tuned gains of its attitude and position controllers; similar controllers have also been proposed in the literature, such as the controller presented in [9]. The inertial parameters of the platform can be easily estimated from the platform’s geometry, and do not depend on the controller gains. Similarly, the attitude controller has a larger authority than the position controller, and thus can be tuned using standard methods, independently of the chosen position controller. The position controller merits further explanation; where as can be seen from Fig. 2 the lateral force limits are coupled with the applied lift force. Since  $\overline{f_{xy}}$  is an estimate of the actual maximum lateral force allowed by the platform for the current lift, it is difficult to chose the best value for this parameter a priori.

Furthermore, the choice of  $\overline{f_{xy}}$  can substantially affect the platform’s performance, where if chosen to be near zero, the platform acts as an underactuated one, with a strong coupling between the position and attitude dynamics. If  $\overline{f_{xy}}$  is chosen to be large enough, the platform behaves as a fully-actuated one, and the position and attitude dynamics become decoupled; special consideration has to be given as not to exceed platform’s physical limits at the applied lift.

In this paper, we aim to study experimentally the above-mentioned interplay between the chosen maximum lateral force and the position dynamics. To be able to systematically tune the controller gains for each chosen  $\overline{f_{xy}}$  value, we use the recently introduced OPTIM-tune algorithm [10]. The method described in [10] performs auto-tuning of controllers, where the tuning is done in cycles, composed of stages where a particular gain is tuned, and the remaining held fixed [10]. The method finds the optimal gains that maximize a designed performance index, and as such can be scaled to tune any of

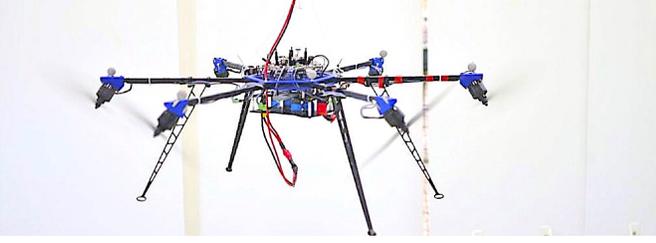


Fig. 1. The considered hexarotor with tilted propellers (Tilt-Hex).

the controller gains affecting the desired performance.

The rest of this letter is structured as follows: Sec. II presents the UAV model and the BLF controller. Sec. III analyzes carefully the parameters of the BLF controller, and discusses the corresponding interdependency. Sec. IV summarizes the tuning algorithm and its use for the tuning of the BLF controller. Finally, Sec. V shows the experimental analysis of the proposed method, and Sec. VI concludes the letter.

## II. MODELING AND CONTROL OF FULLY ACTUATED UAVS WITH BOUNDED LATERAL FORCE

In this section, we briefly describe the modeling and control of fully-actuated aerial vehicles with *Bounded Lateral Force*, in order to introduce the parameters whose automatic tuning represents the goal of the proposed method. A BLF model is a powerful and simple abstraction of several different multi-rotor designs, including, e.g., underactuated, fully-actuated, multi-directional thrust, and thrust vectored designs. For a detailed description of the BLF model and its relation with real multirotor designs, we refer the reader to [8], where all these concepts have been introduced.

We define an inertial world frame  $\mathcal{F}_W$  with origin  $O_W$  and axes  $\{x_W, y_W, z_W\}$ , and a robot frame  $\mathcal{F}_R$ , attached to the vehicle, with origin  $O_R$  and axes  $\{x_R, y_R, z_R\}$ , where  $O_R$  coincides with the Center of Mass (CoM) of the vehicle. We denote by  $p_R \in \mathbb{R}^3$  and  $R_R \in SO(3)$  the position of  $O_R$  in  $\mathcal{F}_W$  and the rotation matrix describing the orientation of  $\mathcal{F}_R$  with respect to (w.r.t.)  $\mathcal{F}_W$ , respectively. The linear velocity of  $O_R$  in  $\mathcal{F}_W$  is denoted with  $v_R \in \mathbb{R}^3$  and the angular velocity of  $\mathcal{F}_R$  w.r.t.  $\mathcal{F}_W$  expressed in  $\mathcal{F}_R$  is denoted with  $\omega_R \in \mathbb{R}^3$ . Finally,  $m_R \in \mathbb{R}_{>0}$  and  $J_R \in \mathbb{R}_{>0}^{3 \times 3}$  denote the vehicle mass and moment of inertia w.r.t. to  $O_R$  in  $\mathcal{F}_R$ , respectively.

Following the Newton-Euler formalism, we can write the equations of motion of this rigid body as

$$\dot{p}_R = v_R, \quad (1)$$

$$\dot{R}_R = R_R[\omega_R]_{\times}, \quad (2)$$

$$m_R \dot{v}_R = -m_R g e_3 + R_R f_R, \quad (3)$$

$$J_R \dot{\omega}_R = -\omega_R \times J_R \omega_R + \tau_R, \quad (4)$$

where  $[\bullet]_{\times}$  is the skew-symmetric operator,  $e_3$  is a unit vector along  $z_W$ ,  $g$  is the gravitational constant,  $f_R = [u_x, u_y, u_z]^T \in \mathcal{U}_f \subset \mathbb{R}^3$  and  $\tau_R \in \mathbb{R}^3$  are the total control force and moment applied on  $O_R$  in  $\mathcal{F}_R$ , respectively, and  $\mathcal{U}_f$  represents the *set of feasible forces* in the robot's frame. A BLF vehicle is characterized by the particular structure of the

set  $\mathcal{U}_f$ , namely,  $\mathcal{U}_f = \mathcal{U}_{xy} \times \mathbb{R}_{\geq 0}$ , where  $\mathcal{U}_{xy}$  is the *set of feasible lateral forces* defined as

$$\mathcal{U}_{xy} = \{[u_x, u_y]^T \in \mathbb{R}^2 \mid u_x^2 + u_y^2 \leq \overline{f_{xy}}^2\}. \quad (5)$$

The distinguishing feature of the BLF model is the presence of the parameter  $\overline{f_{xy}} > 0$ , which represents the *maximum magnitude of lateral (horizontal) force* that the BLF vehicle can produce in  $\mathcal{F}_R$ . As such, and following the allocation strategy from [8], if the required lateral force  $\notin \mathcal{U}_{xy}$ , the controller prioritizes the position controller and tilts the platform differently from the reference attitude trajectory so as to include the lateral force  $\in \mathcal{U}_{xy}$ . The smaller  $\overline{f_{xy}}$ , the closer the BLF vehicle resembles an underactuated multirotor (e.g., a quadrotor) and as a consequence, the more coupled are its lateral motion and attitude dynamics – e.g., a lateral acceleration requires a non-zero tilting of the vehicle. The larger  $\overline{f_{xy}}$ , the more decoupled can be its orientation and lateral motion – e.g., the BLF vehicle can accelerate laterally with a small tilting and can tilt with lateral acceleration close to zero.

There are two main advantages of the BLF model compared to a more accurate multi-parametric and coupled model for the particular multi-rotor aerial vehicle at hand. First, the BLF model is much simpler and requires the identification and use of only one actuation parameter – namely  $\overline{f_{xy}}$ . Second, the BLF model can be made asymptotically stable using a controller (see [8]) that is analytically proven to converge and it has been experimentally demonstrated to effectively stabilize real multirotor platforms modeled as BLF, such as the platform shown in Fig. 1. More accurate models are possible [11], however, they require complex identification procedures of many parameters, which may be impractical. Furthermore, due to their complexity, they can be controlled only resorting to numerical optimization-based control, which typically requires a high computational power that may not be available onboard. Last but not least, such numerical methods do not typically have an analytical guarantee of asymptotical stabilization.

A BLF vehicle can be stabilized along a time-varying and full-pose reference trajectory  $q^r(t) = (p_R^r(t), R_r(t))$  using the analytically proven control law presented in [8], which has the following form:

$$f_R = \text{sat}_{\mathcal{U}_{xy}} \left( (f_r^T R_R e_1) e_1 + (f_r^T R_R e_2) e_2 \right) + (f_r^T R_R e_3) e_3, \quad (6)$$

$$\tau_R = \omega_R \times J_R \omega_R - K_R e_R - K_\omega e_\omega - J_R \left( [\omega_R]_{\times} R_R^T R_d \omega_R^d - R_R^T R_d \dot{\omega}_R^d \right), \quad (7)$$

where

$$f_r = m_R (\dot{v}_R^r + g e_3) - K_p e_p - K_v e_v. \quad (8)$$

Considering the goal of this paper, we omit the details for the sake of compactness and readability, and we refer the reader to [8] for the exact definition of all the terms in the controller as well as its stability proof. In the next section, we focus on the discussion of the parameters used in this control law.

### III. DISCUSSION ON THE CONTROL PARAMETERS AND NEED FOR AUTOMATIC TUNING

The BLF model (1)–(5) has three parameters:  $m_R$ ,  $\mathbf{J}_R$ , and  $\overline{f_{xy}}$ , which are all used in the corresponding controller in (6)–(8) together with the four additional sets of parameters representing the control gain matrices  $\mathbf{K}_p$ ,  $\mathbf{K}_v$ ,  $\mathbf{K}_R$ , and  $\mathbf{K}_\omega$ . In the following, we analyze each parameter from the point of view of interdependency and easiness of identification with methods available in the state of the art.

#### A. Inertial Parameters

The inertial parameters  $m_R$  and  $\mathbf{J}_R$  have clear physical meaning and their offline identification or online estimation is rather straightforward and established (see, e.g., [12] and [13]). Furthermore, their nominal values are typically accurate because they can be retrieved from the CAD model of the system. Therefore, one can safely assume their values to be identifiable with good accuracy using standard methods.

#### B. Gains of the Attitude Control loop

The gain matrices  $\mathbf{K}_R$  and  $\mathbf{K}_\omega$  appear in (7) and affect the attitude dynamics (2),(4), which is independent of the rest of the vehicle dynamics (the position dynamics) and contains only the parameter  $\mathbf{J}_R$ , which, as explained before, can be fairly assumed to be known with good accuracy. Furthermore, the attitude dynamics is fully-actuated and no limits in the control moments appear in the BLF model. As a consequence, the gains  $\mathbf{K}_R$  and  $\mathbf{K}_\omega$  can be easily tuned independently from the other control parameters by using standard PD tuning methods such as the one presented in [14]. Therefore, we can also in this case safely assume that  $\mathbf{K}_R$  and  $\mathbf{K}_\omega$  are tuned with state-of-the-art methods and do not require special attention.

#### C. Maximum magnitude of the Lateral Force

In a real multi-rotor platform, see [15], the maximum magnitude of the lateral force depends on the applied vertical component of the force and the applied full moment. Such lateral bound is typically larger when the requested vertical force exactly compensates for the gravity force and the total moment is zero. The farther the vertical force and the moment are from such two neutral conditions, the smaller the lateral bounds on the horizontal component of the force. Figure 2 shows an example feasible force set at hover and a corresponding BLF model calculated when applying a vertical force opposing gravity, where static hovering has been formally defined in [16].

In the BLF model, on the contrary,  $\overline{f_{xy}}$  is a lumped constant value. If  $\overline{f_{xy}}$  is set too small, then the controller will let the platform behave too close to an underactuated platform, while if  $\overline{f_{xy}}$  is set too large, it could lead to suboptimal behaviors because it may not represent well the lateral bounds induced by the moment and vertical force required by the task.

In conclusion, the parameter  $\overline{f_{xy}}$  plays the role of a ‘lateral-actuation slider’, which position has to be tuned, in order to optimize the behavior of the real controlled platform for the particular task at hand. Such a need calls for an automated tuning algorithm that can optimize the value of  $\overline{f_{xy}}$  based on the controller performance.

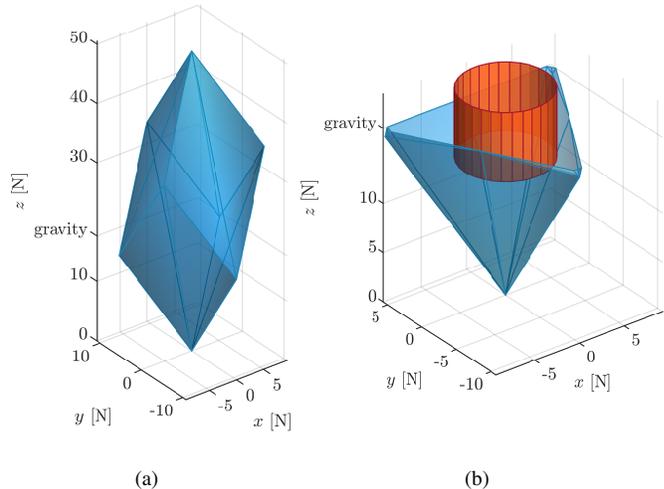


Fig. 2. (a) Feasible force set of the platform in Fig. 1 at hover. (b) The same feasible force set with cross section at the gravity plane. The figure also shows the BLF cylinder for the platform applying a lift force  $\pm 20\%$  of the gravity opposing lift.

#### D. Gains of the Position Control loop

The gain matrices  $\mathbf{K}_p$  and  $\mathbf{K}_v$  appear in (8) and affect the position dynamics (1),(3). It is standard to assume a diagonal structure of  $\mathbf{K}_p$  and  $\mathbf{K}_v$  considering the symmetry of the model and in order to avoid an unnecessary cross-direction coupling induced by the controller. Furthermore, thanks to the horizontal symmetry of the model and controller, one can assume that the first two entries of the diagonals are equal. Therefore, it is reasonable to assume the following structure for  $\mathbf{K}_p$  and  $\mathbf{K}_v$ :

$$\mathbf{K}_p = \text{diag} \{k_p, k_p, k_{p,z}\}, \quad (9)$$

$$\mathbf{K}_v = \text{diag} \{k_v, k_v, k_{v,z}\}. \quad (10)$$

The choice of  $k_{p,z}$  and  $k_{v,z}$  affects the closed-loop vertical dynamics along which the system has a large control authority and is not influenced by the rest of the dynamics. Therefore – similarly to  $\mathbf{K}_R$  and  $\mathbf{K}_\omega$  – the gains  $k_{p,z}$  and  $k_{v,z}$  can be tuned independently, e.g., letting the vehicle move up and down and using standard PD tuning techniques [14].

The remaining parameters, namely  $k_p$  and  $k_v$ , cannot be chosen independently from  $\overline{f_{xy}}$ , because there is a nonlinear saturation on the lateral dynamics that depends on  $\overline{f_{xy}}$  (6). Therefore,  $k_p$  and  $k_v$  have to be chosen in a way that lets the system behave optimally in the lateral motion, while well coping with the saturation induced by  $\overline{f_{xy}}$ .

To provide an insight into the complexity of such an interplay, let us first consider the two extreme cases. If  $\overline{f_{xy}}$  is chosen very small, the platform’s lateral dynamics is in practice underactuated and the platform needs to tilt, to move laterally. Therefore, the gains  $k_p$  and  $k_v$  have to be optimized to let the position dynamics be as fast as possible but ‘slower’ than the attitude dynamics, as in a quadrotor. On the contrary, if  $\overline{f_{xy}}$  is large, the system can move laterally (up to a certain acceleration) without tilting, therefore there is virtually no need to take into account the attitude dynamics in the tuning

of  $k_p$  and  $k_v$ . On the other side, there is still the dynamics of the motor/propeller to consider. In fact, lateral motions without tilting require a much more ample range of propeller spinning velocities compared to the case in which the system moves laterally by tilting (with small  $\overline{f_{xy}}$ ). This phenomenon can be easily appreciated looking at the experiments reported in [8]. Therefore, for large  $\overline{f_{xy}}$ , the dynamics to consider is the motor/propeller one, which has of course different characteristics than the attitude one.

For intermediate values of  $\overline{f_{xy}}$ , a mixture of attitude and motor dynamics influences the optimal choice of the gains  $k_p$  and  $k_v$  in a way that is hard to predict a priori.

## E. Conclusions

From the discussion carried out in this section, it emerges that there are two different types of control parameters in (6)–(8). The first type, namely  $m_R$ ,  $\mathbf{J}_R$ ,  $\mathbf{K}_R$ ,  $\mathbf{K}_\omega$ ,  $k_{p,z}$  and  $k_{v,z}$  can be tuned mostly independently and resorting to state-of-the-art methods such as, e.g., physical parameter identification using least squares approach or PD tuning. The second type, namely  $\overline{f_{xy}}$ ,  $k_p$ , and  $k_v$ , are tightly coupled, and their effects on the system behavior are coupled and nonlinear and one cannot use straightforward methods like PD tuning to tune these parameters. In particular:

- different values of  $\overline{f_{xy}}$  may be chosen depending on the motion task at hand, where there is no clear 'best value' until the task is specified;
- for each value of  $\overline{f_{xy}}$  it is expected to obtain different optimized values for  $k_p$  and  $k_v$ , due to the nonlinear interplay explained before

Therefore, in the remaining of this paper, for the first time in the literature, we focus our attention on the *optimal tuning of  $k_p$  and  $k_v$  for different values of  $\overline{f_{xy}}$  in real platforms modeled and controlled as BLF*. First, we describe the automatic method used for the tuning (Sec. IV), and then we test the presented method on a real platform. These tests demonstrate the existence of the expected dependency as well as the improvement of the controller performance following the presented tuning method (Sec. V).

*Note that the stability of the controller against unmodeled and external disturbances following the choice of  $k_p$  and  $k_v$  has been thoroughly studied in Appendix B.*

## IV. DATA-BASED PARAMETER TUNING ALGORITHM

The optimization algorithm that is used to tune the gains  $k_p$  and  $k_v$ , for a given value of the parameter  $\overline{f_{xy}}$ , is an instantiation of the model-free OPTIM-tune algorithm presented in [10] and requires only a measurable metric of the performance of the controller in order to work. Further analysis on the convergence of the OPTIM-tune algorithm can be found in Appendix A. The overall method is a combination of two nested loops: *i*) An outer loop, described in Algorithm 1, and *ii*) an inner loop, also called *single parameter tuning*, described in Algorithm 2.

Algorithm 1 (the outer loop) receives as input: *i*) the maximum lateral force  $\overline{f_{xy}}$  (which is kept constant during the tuning), *ii*) the two sets  $\mathcal{D}_{k_p}^{(1)}$ ,  $\mathcal{D}_{k_v}^{(1)}$  that represent the intervals

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**Algorithm 1:** Tuning of controller gains  $k_p$  and  $k_v$  for a fixed  $\overline{f_{xy}}$  value.

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**Data:**  $\overline{f_{xy}}$ ,  $\mathcal{D}_{k_p}^{(1)}$ ,  $\mathcal{D}_{k_v}^{(1)}$ ,  $N$  and  $N_b$   
**Result:** optimized  $k_p^* \in \mathcal{D}_{k_p}^{(1)}$ ,  $k_v^* \in \mathcal{D}_{k_v}^{(1)}$   
 $i \leftarrow 1$ ;  
**while**  $i \leq N_b$  **do**  
     $k_v \leftarrow \overline{\mathcal{D}_{k_v}^{(i)}}$ ;  
     $\mathcal{D}_{k_p}^{(i+1)} \leftarrow \text{Algorithm}_2(\mathcal{D}^{(1)} = \mathcal{D}_{k_p}^{(i)}, \xi = k_v, \overline{f_{xy}}, N)$ ;  
     $k_p \leftarrow \overline{\mathcal{D}_{k_p}^{(i+1)}}$ ;  
     $\mathcal{D}_{k_v}^{(i+1)} \leftarrow \text{Algorithm}_2(\mathcal{D}^{(1)} = \mathcal{D}_{k_v}^{(i)}, \xi = k_p, \overline{f_{xy}}, N)$ ;  
     $i \leftarrow i + 1$   
**end**  
 $k_p^* = \overline{\mathcal{D}_{k_p}^{(N_b)}}$ ;  
 $k_v^* = \overline{\mathcal{D}_{k_v}^{(N_b)}}$ ;

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**Algorithm 2:** Generic single parameter tuning.

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**Data:**  $\mathcal{D}^{(1)} = [-\theta^{(1)}, +\theta^{(1)}]$ ,  $\xi$ ,  $\overline{f_{xy}}$ , and  $N$   
**Result:**  $\mathcal{D}^{(N)} = [-\theta^{(N)}, +\theta^{(N)}]$   
 $i \leftarrow 1$ ;  
**while**  $i \leq N$  **do**  
    calculate contraction factor  $\rho_i$ ;  
    calculate candidate ( $i + 1$ ) bounds:  
         $-\hat{\theta}^{(i+1)} = -\theta^{(i)} + \rho_i(+\theta^{(i)} - -\theta^{(i)})$   
         $+\hat{\theta}^{(i+1)} = -\theta^{(i)} + (1 - \rho_i)(+\theta^{(i)} - -\theta^{(i)})$   
    execute flight test with params  $\xi$ ,  $-\hat{\theta}^{(i+1)}$ , and  $\overline{f_{xy}}$ ;  
     $-f \leftarrow f(\mathbf{s}_e)$ ;  
    execute flight test with params  $\xi$ ,  $+\hat{\theta}^{(i+1)}$ , and  $\overline{f_{xy}}$ ;  
     $+f \leftarrow f(\mathbf{s}_e)$ ;  
    **if**  $-f < +f$  **then**  
         $\mathcal{D}^{(i+1)} \leftarrow [-\hat{\theta}^{(i+1)}, +\theta^{(i)}]$ ;  
    **else**  
         $\mathcal{D}^{(i+1)} \leftarrow [-\theta^{(i)}, +\hat{\theta}^{(i+1)}]$ ;  
    **end**  
     $i \leftarrow i + 1$ ;  
**end**

---

over which the gains  $k_p$  and  $k_v$  are optimized, and *iii*) two integers,  $N_b$  and  $N$ , representing the number of iterations in the outer and inner loop, respectively.

Algorithm 2 (inner loop, or single parameter tuning) receives as input: *i*)  $\overline{f_{xy}}$  and  $N$  (the same of Algorithm 1), *ii*) the starting set, denoted with  $\mathcal{D}^{(0)}$ , in which one of the two gains (either  $k_p$  or  $k_v$ ) will be optimized, and *iii*) the value of the other parameter (either  $k_v$  or  $k_p$ ) that is kept fixed during the execution of Algorithm 2, denoted with  $\xi$ . The algorithm provides as output a new set  $\mathcal{D}^{(N)}$ , which is a contraction of  $\mathcal{D}^{(0)}$  and is guaranteed to contain the optimum value of the corresponding parameter.

Algorithm 1 executes  $N_b$  times a basic iteration, in which two instances of Algorithm 2 are performed sequentially to contract  $\mathcal{D}_{k_p}^{(i)}$  and  $\mathcal{D}_{k_v}^{(i)}$ . In the first instance,  $k_v$  is kept fixed at its current estimate and the set to which the optimal  $k_p$  belongs is contracted, thus generating an improved estimate of  $k_p$ . In the second instance – symmetrically – the new estimate of  $k_p$  is kept fixed while the set to which the optimal  $k_v$  belongs

is contracted, thus generating a new improved estimate of  $k_v$ . At the end of Algorithm 1 the optimized values of the gains are returned in the form of the mid-values of the intervals generated by the contractions of the last ( $N_b$ -th) iteration, i.e.,  $\mathcal{D}_{k_p}^{(N_b)}$  and  $\mathcal{D}_{k_v}^{(N_b)}$ . Such mid-values are denoted with  $\overline{\mathcal{D}}_{k_p}^{(N_b)}$  and  $\overline{\mathcal{D}}_{k_v}^{(N_b)}$ , respectively.

Algorithm 2 performs the set contraction implementing  $N$  smaller consecutive contraction steps. Each step executes two flight tests with the vehicle, using  $\overline{f_{xy}}$  as lateral force bound and  $\xi$  as the temporarily fixed gain. The goal of each flight test is to evaluate the effect of a new candidate for the upper and lower bound of the set containing the gain to be optimized. A new upper bound candidate  $+\hat{\theta}^{(i+1)}$  is tested in the first flight test, while a new lower bound candidate  $-\hat{\theta}^{(i+1)}$  is tested in the second one. Each flight test is followed by the evaluation of a cost function  $f$  that depends on the state error  $s_e$ , i.e., the vector describing the error between the desired and the measured state of the system during the execution of a flight test. The candidate bound that corresponds to the test which returned the lower value of  $f$  is used as new upper or lower bound for the set of the estimated parameter, thus producing the sought contraction for the particular step. This process is repeated  $N$  times. The last obtained set  $\mathcal{D}^{(N)}$  is returned as the result of the algorithm.

In our specific case, the goal is to find the optimal controller gains that ameliorate the lateral trajectory tracking while the platform remains as much as possible horizontal – thus exploiting at best the lateral force capability of BLF platforms. In line with such goal, and assuming that the position trajectory of a flight test is composed by  $N_c$  reference points, the corresponding cost function  $f$  is defined as follows:

$$f(s_e) = \underbrace{\sum_{k=1}^{N_c} |e_k|}_{f_e} + \frac{1}{Q} \underbrace{\sum_{k=1}^{N_c} |\phi_k|}_{f_\phi} = \sum_{k=1}^{N_c} J, \quad (11)$$

where  $|e_k|$  is the norm of the lateral position error,  $|\phi_k|$  is the norm of the tilt angular error and  $J$  is the weighted error per reference point. The parameter  $Q$  is used to weight the contribution of the angular error in the overall performance index, giving more or less importance to the fact that the platform remains horizontal while following the position trajectory.

## V. TEST OF TUNING ON A REAL PLATFORM AND VALIDATION OF THE INTERDEPENDENCY HYPOTHESIS

We validate the proposed approach with the BLF platform presented in [8]; the platform is referred to as the Tilt-Hex and is shown in Fig. 1. The platform is a hexarotor constructed from six 12" tilted propellers equally-spaced about the platform center of mass. The platform has a mass of 1.8 kg, and an inertia tensor  $\mathbf{J}_R = \text{diag}\{11.5, 11.4, 19.4\} \cdot 10^{-6} \text{ kg} \cdot \text{m}^2$ .

In addition, the platform is endowed with an Inertial Measurement Unit (IMU) providing acceleration and angular velocity measurements at 1 kHz, and is tracked with a motion capture system at 100 Hz. Both measurements are fused with an Unscented Kalman Filter running at 1 kHz, providing an

estimate of the platform state. The motion controller runs on-board at 1 kHz, and brushless motor controllers (BLDC ESC) regulate the propellers' speed using an in-house developed closed-loop speed controller [17]. Most of the software is developed in C++ and runs on an on-board PC, with the exception of the gain tuning algorithm which runs in MATLAB/Simulink on a ground PC. Most of the onboard software are open source, and can be found at <https://git.openrobots.org/projects/telekyb3>, while the OPTIM-tune software is available at <https://github.com/AppliedControlTechniques/Optim-tune>.

A simulation and an experimental campaign have been carried out, in which the task has been to tune the controller gains, for the selected values of  $\overline{f_{xy}}$  and different  $Q$  ratios. The interested reader is referred to the multimedia attachment of this letter for the experiments' videos. In all the experiments and simulations, the lateral acceleration of the reference trajectory has been pushed up to  $1.5 \text{ ms}^{-2}$ , jerk to  $10 \text{ ms}^{-3}$ , lateral velocity to  $2 \text{ ms}^{-1}$ , to bring the platform to its lateral motion limits. During each of the tuning experiments, the initial domains of the controller gains have been chosen such that  $\mathcal{D}_{k_p}^{(1)} = [10, 30]$  and  $\mathcal{D}_{k_v}^{(1)} = [5, 15]$ , and each domain has been contracted  $N = 12$  times by the inner loop in each of the  $N_b = 2$  outer loop iterations.

Finally, to stress the lateral position tracking and horizontality of the platform (zero tilt), the reference trajectory has been chosen as a back and forth path parallel to  $x_W$ .

### A. Simulative Analysis of the Cost Function Landscape

To study the cost function landscape, we simulated the above mentioned platform with the corresponding controller, and computed the different components of the cost function while varying the controller positional gains  $k_p$  and  $k_v$  over a discretized grid of  $\mathcal{D}_{k_p}^{(1)}, \mathcal{D}_{k_v}^{(1)}$ .

Fig. 3 shows the contour plots of the two components of  $f(s_e)$  at  $\overline{f_{xy}} = 2$  and  $\overline{f_{xy}} = 8$ . The functions  $f_e$  and  $f_\phi$  are shown separately to understand the effect of each on the performance of the presented tuning scheme. This figure shows that as  $\overline{f_{xy}}$  is increased from 2 to 8, the lateral position error component  $f_e$  decreases slightly for almost all values in the given range, and the number of local minima slightly increases as the position changes moderately. On the other hand,  $f_\phi$  shows a different behavior, where it can be seen that the number of local minima increases substantially for  $\overline{f_{xy}} = 8$ . In addition, as  $\overline{f_{xy}}$  is increased, the values of  $f_\phi$  decrease substantially; this can be seen from the different scales of the corresponding contour plots. As such, as  $\overline{f_{xy}}$  is increased the effect of the angular component on the overall cost function decreases; this is similar to an increase in the  $Q$  value. This shows the decreased effect of  $f_\phi$  (and correspondingly the angular dynamics) on the tuning of the position controller gains as  $\overline{f_{xy}}$  increases, even when  $Q$  is kept constant.

### B. Experimental Test of the Tuning Algorithm

We conduct an experimental campaign to demonstrate the tuning of the proposed algorithm and to show the relation between the chosen  $\overline{f_{xy}}$  and the optimized controller gains.

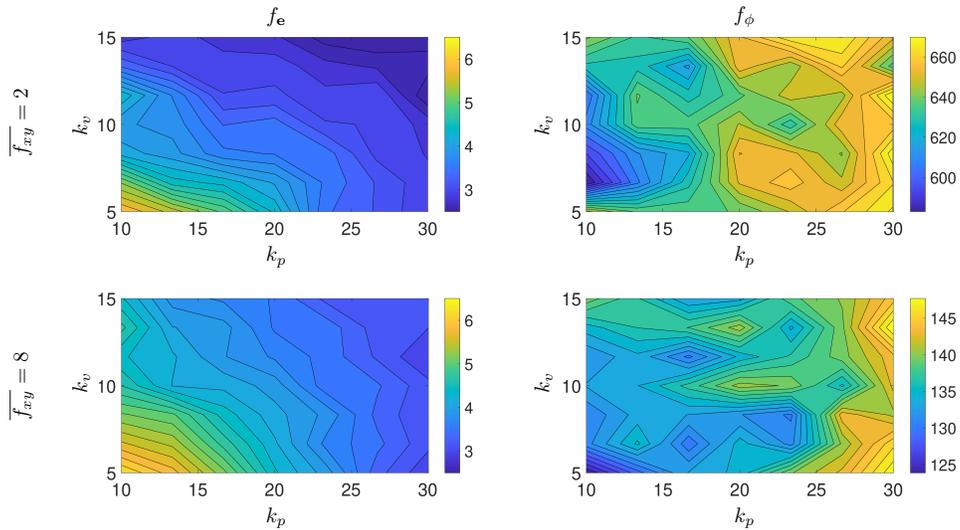


Fig. 3. Contour plots of  $f_e(k_p, k_v)$  and  $f_\phi(k_p, k_v)$  for two different values of  $\overline{f_{xy}}$  computed via simulation on a discretization of the ranges  $\mathcal{D}_{k_p}^{(1)}, \mathcal{D}_{k_v}^{(1)}$ .

Fig. 4 shows the average of the optimized controller gains and standard deviation of each for different values of  $\overline{f_{xy}}$  and  $Q$ , where the optimization at each  $\overline{f_{xy}}$  and  $Q$  value have been repeated thrice. This figure shows that as  $\overline{f_{xy}}$  increases, the optimized  $k_p$  increases and the optimized  $k_v$  decreases for both  $Q$  values. In the case of  $Q = 300$ , the contribution of  $f_\phi$  is reduced substantially leading to higher optimized  $k_p$  and  $k_v$  values as the optimization allows for more aggressive position tracking without taking the angular tracking error into consideration.

In order to get an improved understanding of how the algorithm operates, we show the evolution of the optimization for two of the above optimization cases, corresponding to  $\overline{f_{xy}} \in \{2, 8\}$  at a constant  $Q = 50$ . Fig. 5 shows the corresponding evolution for  $\overline{f_{xy}} = 2$  and  $\overline{f_{xy}} = 8$ . This figure shows the evolution of the controller gains and the resulting performance index for each of the two  $\overline{f_{xy}}$  values. These experiments clearly show the improved trajectory tracking of the controller with the optimized gains, and that while the initial controller gains are the same for both experiments, the optimized gains vary substantially for different values of  $\overline{f_{xy}}$ , supporting the interdependence of such parameters that is claimed in this work.

## VI. CONCLUSIONS

In this letter, we studied the interplay between the parameters and gains of the BLF platform while controlled via a full-position controller. We showed that the optimally chosen position control gains rely largely on the estimated lateral force limit.

While one can fix the estimated lateral force limit at one of its extreme values, we discussed the effect of these extremes, where if the limit is chosen small enough the platform behaves as an underactuated one, while if chosen large enough, it might exceed the physical capabilities of the platform, and thus the controller could behave sub-optimally.

We then presented a detailed method for the auto-tuning of the position control gains for different estimated lateral

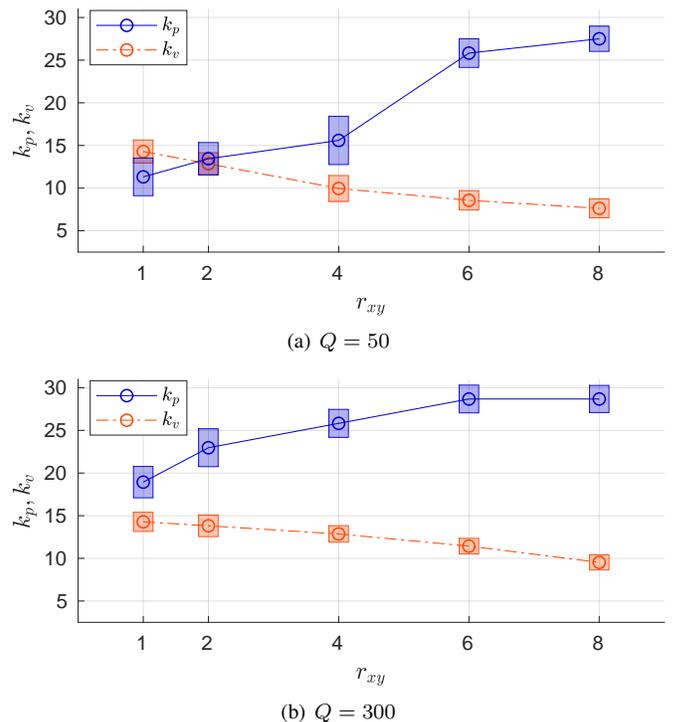


Fig. 4. Optimized gains ( $k_p, k_v$ ) of the position controller versus  $\overline{f_{xy}}$  for two values of the weighting ratio  $Q$ . For each optimized gains, we show the average value and the standard deviation bar computed from three repetitions of each experiment.

force limits, and showed how these optimized gains vary accordingly. However, the choice of this parameter is still an open question, where for each application, different values of the lateral force limit should be chosen. Moreover, based on the applied lift force, this parameter has to be changed with the corresponding optimized gains. As such, in the future, we propose to apply a gain scheduling approach to choose online the ‘best’ value of  $\overline{f_{xy}}$  and the corresponding optimized gains.

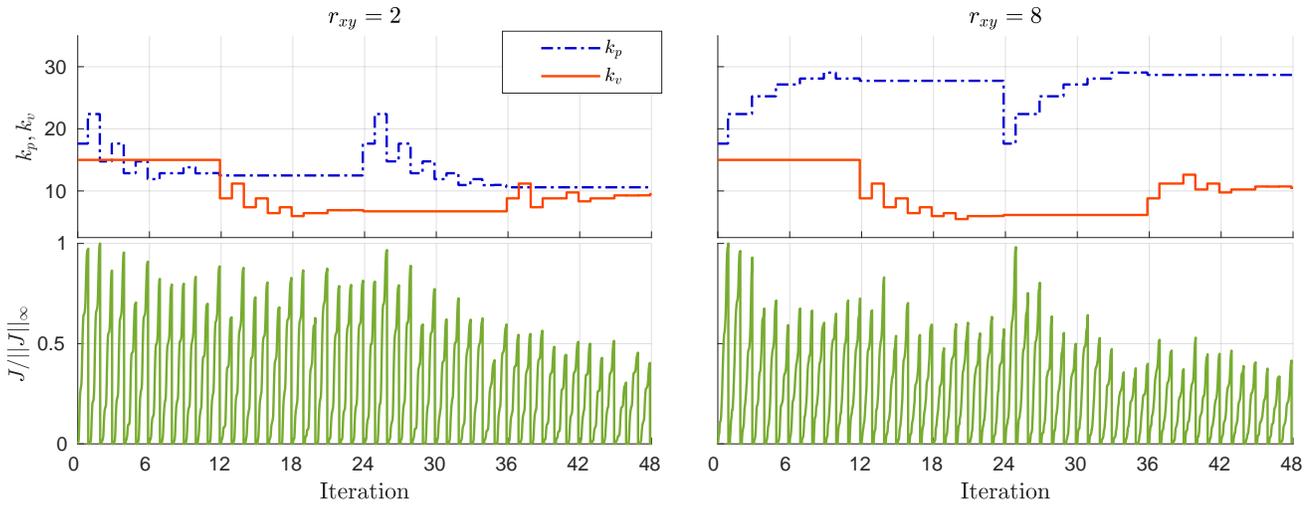


Fig. 5. Evolution of the gain tuning algorithm for  $\overline{f_{xy}} = 2$  and  $\overline{f_{xy}} = 8$  while  $Q = 50$  showing the controller gains  $k_p$ ,  $k_v$ , and the cost function  $f(\mathbf{s}_e)$  for  $N = 12$  and  $N_b = 2$ .

#### ACKNOWLEDGMENT

We thank Anthony Mallet and Dario Sanalidro for their help in the hardware and software implementation of the experimental tests.

## APPENDIX A

## DISCUSSION OF OPTIMIZATION ISSUES IN OPTIM-TUNE

The OPTIM-tune method is based on zero-order one-dimensional search, as in the case of Piyavskii's algorithm [18], or of Timonov's [19] and Shoen's [20], formed here in sequences named bootstraps. The OPTIM-tune algorithm assumes that  $f$  (the cost function) is a continuous function, with a finite number of local minima in its search interval  $\mathcal{D}_j^{(i)}$ . We do note that the zero-order one-dimensional search requires  $f$  to hold a strongly quasiconvex property which we will prove to hold at least in the subintervals of the search interval. Note that the proof the following propositions (1-3) can be found in [21].

*Proposition 1.* Let  $f$  be a continuous function over an interval  $[a, b]$  with a finite number of locally optimal minima. Then there exists a strictly positive number  $p$  such that  $f$  is either strongly quasiconvex or strongly quasiconcave over any  $[q, q + p]$ , with  $q \in [a, b - p]$ .

*Proposition 2.* Let  $f$  be a continuous function over an interval  $[a, b]$ . If there exists a finite number of locally optimal minima, then there exists a strictly positive number  $p$  such that  $f$  is either strongly quasiconvex or strongly quasiconcave over any  $[q, q + p]$ , with  $q \in [a, b - p]$ .

*Corollary 1.* Following Proposition 1 and 2, it is guaranteed that  $f$  is strongly quasiconvex over the contracted subintervals of its search domain, even if it was not strongly quasiconvex over its initial search domain.

In what follows, we aim at providing a proof that globally  $\varepsilon$ -optimal points of strongly quasiconvex functions can be efficiently found by zero-order optimization algorithms.

*Proposition 3.* Let  $f$  be a continuous strongly quasiconvex function over an interval  $[a, b]$ . Let  $\phi_l$  ( $l = 1, \dots, 4$ ) denote four evaluation points,  $\phi_j^{(i)} < \phi_{j+1}^{(i)}$  at the  $i$ -th iteration of the Fibonacci search. In this case, the one-dimensional search fails to detect whether  $f$  is strongly quasiconvex or not if and only if  $f(\phi_1^{(i)}) \leq f(\phi_3^{(i)}) \leq f(\phi_4^{(i)}) \leq f(\phi_2^{(i)})$  for all  $i$  or,  $f(\phi_1^{(i)}) \geq f(\phi_3^{(i)}) \geq f(\phi_4^{(i)}) \geq f(\phi_2^{(i)})$  for all  $i$ .

*Corollary 2.* From Proposition 3 it holds that if the four considered points form neither a monotone increasing, nor monotone decreasing sequence, an  $\varepsilon$ -optimal solution inside a subinterval is guaranteed to exist.

The theorem given in [22] states that if  $f$  is a Lipschitz function with constant  $L$  over the interval  $[a, b]$ , and  $f$  is assumed to have a finite number of locally optimal points, then there exists a tolerance  $\varepsilon$ , upper bounded by the distance between two locally optimal points, which guarantees  $\varepsilon$ -convergence. The Fibonacci algorithm used here guarantees that a globally  $\varepsilon$ -optimal solution is found within  $\mathcal{D}_j^{(i)}$  for an unimodal function. The OPTIM-tune method is, in addition, insensitive to such common problems as flat nature of the  $f$  function over an interval, where other algorithms might fail (such as Piyavskii's or Hansen, Jaumard and Lu's algorithms [23]).

Now let us consider the multivariate property of  $f$ . Let  $f(\theta_1, \theta_2)$  be a unimodal function within acceptable ranges of its parameters. For arbitrary  $a^{(i)} \in \mathcal{D}_1^{(i)}$  and  $b^{(i)} \in \mathcal{D}_2^{(i)}$ , the

functions  $f(\theta_1, b^{(i)})$  and  $f(a^{(i)}, \theta_2)$  are unimodal, and can be denoted as  $\psi(\theta)$  with a corresponding range  $[\alpha, \beta]$ .

*Proposition 4.* The contracted range of  $\psi(\theta)$  always contains its minimum.

*Proof.* Let us denote by  $\theta^*$  as the minimum of  $\psi(\theta)$  in the range  $[\alpha, \beta]$ , and by  $\theta^{(1)}, \theta^{(2)}$  as two arbitrary variables such that  $\alpha \leq \theta^{(1)} \leq \theta^{(2)} \leq \beta$ .

$$\text{if } \alpha \leq \theta^{(1)} < \theta^{(2)} \leq \theta^* \text{ then } \psi(\theta^{(1)}) > \psi(\theta^{(2)}) \quad (12)$$

$$\text{and if } \theta^* \leq \theta^{(1)} < \theta^{(2)} \leq \beta \text{ then } \psi(\theta^{(1)}) < \psi(\theta^{(2)}) \quad (13)$$

Then by contracting the range of  $\psi(\theta)$  following Alg.2,  $\theta^*$  will always be contained in the contracted range.  $\square$

*Proposition 5.* OPTIM-tune has a natural escaping capability (see [24] for definition), when reduction of the subinterval takes place (Alg.2).

*Proof.* It was proven in [25] that zero-order algorithms avoid strict saddle points and guarantee convergence to local optima. Moreover, In Alg.1 the escaping capability is further ensured due to bootstrapping and altering optimization variables, to enable finding an improved minima.  $\square$

As such, and following the above propositions, the OPTIM-tune method finds the  $\varepsilon$ -optimal point in the original search domain by contracting the mentioned interval into subintervals. The contractions guarantee that the  $\varepsilon$ -optimal will always be contained in the corresponding subinterval, while ensuring the escaping capability. Finally, the presented algorithm contracts these intervals in an exponential manner, while altering the optimization variables, and as such, finding an  $\varepsilon$ -optimum in a short number of iterations.

## APPENDIX B

## ROBUSTNESS ISSUES FOR A LINEARIZED MODEL

## A. Preliminaries

While the OPTIM-tune method does not require any knowledge of the UAV model, it does require an initial range of gains in which it tries to find the optimal value, and in which the controller is assumed to perform in a stable manner. These may result either from initial test flights, or through the analysis of simulation-based flights. In what follows, we assess the stability of the used controller [8] against external disturbances.

First, let us linearize the system around the working points from the problem above. To do so, let us define the following notations:

notation	definition	dimension
$\mathbf{R}$	Rotation matrix of $\mathcal{F}_R$ in $\mathcal{F}_W$	$\mathbb{R}^{3 \times 3}$
$\theta$	rotation angle about $\mathbf{y}_W$	$\mathbb{R}$
$\theta_d$	desired rotation angle about $\mathbf{y}_W$	$\mathbb{R}$
$e_p$	position error in $\mathcal{F}_W$	$\mathbb{R}^3$
$e_v$	linear velocity error in $\mathcal{F}_W$	$\mathbb{R}^3$
$e_R$	angular error in $\mathcal{F}_W$	$\mathbb{R}^3$
$e_\omega$	angular velocity error in $\mathcal{F}_R$	$\mathbb{R}^3$
$\mathbf{J}_R$	Inertia Matrix in $\mathcal{F}_R$	$\mathbb{R}^{3 \times 3}$
$e_{px}$	position error in $x_R$	$\mathbb{R}$
$e_{vx}$	linear velocity error in $x_R$	$\mathbb{R}$
$\mathbf{K}_p$	position gain matrix	$\mathbb{R}^{3 \times 3}$
$\mathbf{K}_v$	linear velocity gain matrix	$\mathbb{R}^{3 \times 3}$
$\mathbf{K}_\omega$	angular velocity gain matrix	$\mathbb{R}^{3 \times 3}$
$\mathbf{K}_R$	angular gain matrix	$\mathbb{R}^{3 \times 3}$

Further, we assume all gain matrices to be diagonal matrices as follows:  $\mathbf{K}_p = \text{diag}(k_p, k_p, k_{pz})$ ,  $\mathbf{K}_v = \text{diag}(k_v, k_v, k_{vz})$ ,  $\mathbf{K}_\omega = \text{diag}(k_{\omega x}, k_{\omega y}, k_{\omega z})$ , and  $\mathbf{K}_R = \text{diag}(k_{Rx}, k_{Ry}, k_{Rz})$ . Similarly, we assume the inertia matrix to be a diagonal matrix such that  $\mathbf{J}_R = \text{diag}(J_{11}, J_{22}, J_{33})$ . Finally,  $\mathbf{f}_r$  is divided about the different axes of  $\mathcal{F}_W$  axes such as  $\mathbf{f}_r = \{f_{rx}, f_{ry}, f_{rz}\}$  and  $\omega$  is divided about the different axes of  $\mathcal{F}_R$  such as  $\omega = \{\omega_x, \omega_y, \omega_z\}$ .

### B. Mass uncertainty

In the following formulation, we assume a constant reference consisting of a motion along the  $x$  axis, and rotation about the roll angle  $\theta$ , while assuming the motion and rotation about all other axes to be negligible, and  $\mathbf{R}_r = \mathbf{I}$ . Following the above assumptions, the attitude controller boils down to the stabilization of the following error term:

$$e_\theta = \theta - \theta_d, \quad (14)$$

geometrically, to check if  $\mathbf{f}_r \in \mathcal{U}_{xy}$  (condition (33) from [8]) it is enough to check the following:

$$\text{find } \theta \text{ such that } \mathbf{R}_y(\theta) \mathbf{f}_r \mathbf{e}_1 \leq \overline{f_{xy}}. \quad (15)$$

If (15) holds,  $\theta_d = 0$  and  $e_\theta = \theta$ , otherwise, the optimization problem from [8] will reduce to finding an equality from (15), as such:

$$\mathbf{R}_y(\theta_d) \mathbf{f}_r \mathbf{e}_1 = \overline{f_{xy}}, \quad (16)$$

$$f_{rx} \cos(\theta_d) + f_{rz} \sin(\theta_d) = \overline{f_{xy}}, \quad (17)$$

$$\theta_d = 2 \tan^{-1} \left( \frac{f_{rz} \pm \sqrt{\|\mathbf{f}_r\|^2 - \overline{f_{xy}}^2}}{f_{rx} + \overline{f_{xy}}} \right). \quad (18)$$

Let  $f_{rx} = -k_p e_{px} - k_v e_{vx}$  and  $f_{rz} = mg - k_{pz} e_{pz} - k_{vz} e_{vz}$  (as previously defined). By assuming that  $k_{pz} e_{pz} + k_{vz} e_{vz}$  to be negligible compared to  $mg$ , we can simplify  $f_{rz}$  such as  $f_{rz} = mg$ . As such

$$\theta_d = 2 \tan^{-1} \left( \frac{mg \pm \sqrt{(mg)^2 + k_p^2 e_{px}^2 + k_v^2 e_{vx}^2 - \overline{f_{xy}}^2}}{-k_p e_{px} - k_v e_{vx} + \overline{f_{xy}}} \right). \quad (19)$$

A regulation task is considered, thus to have stable hovering it must hold that  $e_{px} = p_x - p_{dx} = p_x$ ,  $e_{vx} = v_x - v_{dx} = v_x$ , and  $\theta_d$  should be expanded around  $p_x = 0$ ,  $v_x = 0$ , to obtain

$$\hat{\theta}_d = a + b_1 p_x + b_2 v_x. \quad (20)$$

Thus, the plant and controller state space model can be written as follows:

$$\dot{\mathbf{p}} = \mathbf{v}, \quad (21)$$

$$\dot{\mathbf{v}} = -\frac{1}{m} \mathbf{R} \mathbf{K}_p \mathbf{p}_x - \frac{1}{m} \mathbf{R} \mathbf{K}_v \mathbf{v}_x, \quad (22)$$

$$\dot{\mathbf{R}} = \mathbf{R} [\boldsymbol{\omega}]_X, \quad (23)$$

$$\dot{\boldsymbol{\omega}} = -\mathbf{J}_R^{-1} \mathbf{K}_R \begin{bmatrix} 0 \\ \theta - a - b_1 p_x - b_2 v_x \\ 0 \end{bmatrix} + \mathbf{J}_R^{-1} \mathbf{K}_\omega \boldsymbol{\omega}, \quad (24)$$

where  $a$ ,  $b_1$  and  $b_2$  result from a Taylor series expansion. Following the previously stated assumptions,  $\mathbf{R} = \mathbf{R}_y(\theta)$ ,

$$\mathbf{R} = \begin{bmatrix} \cos(\theta) & 0 & \sin(\theta) \\ 0 & 1 & 0 \\ -\sin(\theta) & 0 & \cos(\theta) \end{bmatrix} = \begin{bmatrix} r_1 & 0 & r_2 \\ 0 & 1 & 0 \\ -r_2 & 0 & r_1 \end{bmatrix}. \quad (25)$$

In addition to the previous assumptions, let us assume that the platform is near hover. Following the above assumptions,  $\theta$  and  $\omega_2$  are assume to be near zero, while  $\omega_1$  and  $\omega_3$  are considered negligible. As such,  $r_1 \approx 1$ ,  $r_2 \approx \theta$ ,  $\omega_1 = \omega_3 = r_1 \omega_3 = 0$ . Moreover, we can assume  $p_y = v_y = v_z = 0$ , and  $e_z = e_y = 0$ . As such, the state space model of the plant and controller can be reduced to the following:

$$\dot{p}_x = v_x, \quad (26)$$

$$\dot{v}_x = -\frac{1}{m} (k_p p_x + k_v v_x), \quad (27)$$

$$\dot{v}_z = 0, \quad (28)$$

$$\dot{r}_1 = 0, \quad (29)$$

$$\dot{r}_2 = \omega_2, \quad (30)$$

$$\dot{\omega}_1 = -\frac{k_{\omega x}}{J_{11}} \omega_1, \quad (31)$$

$$\dot{\omega}_2 = -\frac{k_{\omega y}}{J_{22}} \omega_2 + \frac{k_{Rx}}{J_{22}} (a + b_1 p_x + b_2 v_x - \theta), \quad (32)$$

$$\dot{\omega}_3 = -\frac{k_{\omega z}}{J_{33}} \omega_3. \quad (33)$$

Following the above linearization of the system, we introduce a parametric uncertainty in the mass of the UAV; the uncertainty is modeled in  $m$  such as  $\frac{1}{m} = \frac{1}{m^\circ + p_m \delta_m}$ , where  $m^\circ$  is a nominal mass,  $p_m$  defines a range for uncertain mass and  $\delta_m$  is an uncertainty of unknown magnitude. As all the mathematical models of physical system suffer from inaccuracies, which might also result from non-exact measurements of no possibility to identify the phenomena which govern complex dynamical systems, let a simple model plus uncertainty of be considered. The derivation will consider  $x$  axis terms

In order to test the robust stability, the influence that the considered uncertainty can exert on the relations in the system

is examined. Let the two signals be introduced,  $u_\Delta$  as the input to the uncertainty, and  $y_\Delta$  as the output,

$$\dot{p}_x = v_x, \quad (34)$$

$$\dot{v}_x = u_\Delta, \quad (35)$$

$$u_\Delta = -\frac{1}{m^\circ} ((K_{P11}p_x + k_v v_x) - y_\Delta), \quad (36)$$

$$y_\Delta = m^\circ \delta_m u_\Delta. \quad (37)$$

Having assumed zero initial conditions, and doing the Laplace transform ( $K_{P11} = k_p, k_v = k_v$ )

$$M(s) = \frac{U_\Delta(s)}{Y_\Delta(s)} = \frac{-s^2}{m^\circ s^2 + k_v s + k_p} \quad (38)$$

As

$$\frac{1}{1 - M(s)m^\circ \delta_m} = \frac{m^\circ s^2 + k_v s + k_p}{m^\circ (1 + \delta_m) s^2 + k_v s + k_p} \quad (39)$$

is proper, we concur there is no destabilizing uncertainty, as per Routh stability criterion  $m^\circ (1 + \delta_m) > 0, k_p > 0, k_v > 0$  at all times.

Since [26]  $1 - M(s)m^\circ \delta_m$  has a proper and stable inverse for all  $\delta_m \in \Delta$ , where  $\Delta$  describes the uncertainty set, then the controller robustly stabilizes the considered uncertain plant against  $\Delta$ . Equivalently, since there exists  $r$  such that  $\|M\|_\infty \leq \frac{1}{r}$  holds, with  $|m^\circ \delta_m| < r$  and  $\delta_m \in \Delta$ , then  $1 - M(s)m^\circ \delta_m$  has a proper and stable inverse for all  $\delta_m \in \Delta$ , as per small gain theorem (see Fig. 6, with  $r = 1, m = 1, k_p = 20, k_v = 10$ ).

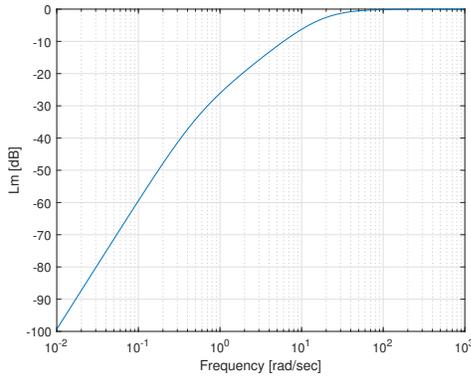


Fig. 6. Log-magnitude plot of  $M$  (vide small-gain theorem)

For larger values of  $k_p$  and  $k_v$  (which define the first two diagonal terms in  $\mathbf{K}_P$  and  $\mathbf{K}_V$ ) the left part of the plot moves up, which means we have lower capability to reject possible low-frequency uncertainty, what forms a natural bound expressed by reducing robustness against such uncertainties. As the range of gains remains unknown, there is a need to optimize them for a given  $f_{xy}$ . In addition, as the method allows one to feed a reference primitive with selected dynamics, OPTIM-tune can find optimized controller gains to improve tracking. Shaping the loop to have narrow-bandwidth like properties, resembles the results obtained from robustness issues optimization.

### C. Sensitivity vs. complementary sensitivity

On the basis of the first-order Taylor expansion of (18) at  $e_{x0} = e_{x0} = 0$  one gets

$$a = 2 \tan^{-1} \left( \frac{mg \pm \sqrt{(mg)^2 - f_{xy}^2}}{f_{xy}} \right), \quad (40)$$

$$b_1 := \frac{2k_p \left( mg \pm \sqrt{(mg)^2 - f_{xy}^2} \right)}{f_{xy}^2 + \left( mg \pm \sqrt{(mg)^2 - f_{xy}^2} \right)^2}, \quad (41)$$

$$= \frac{k_p}{mg} \quad (42)$$

$$b_2 := \frac{2k_v \left( mg \pm \sqrt{(mg)^2 - f_{xy}^2} \right)}{f_{xy}^2 + \left( mg \pm \sqrt{(mg)^2 - f_{xy}^2} \right)^2}, \quad (43)$$

$$= \frac{k_v}{mg}. \quad (44)$$

Now, having assumed that  $\theta \approx 0$  we get  $\sin(\theta) \approx \theta, \cos(\theta) \approx 1$ , and by additionally assuming  $\omega_2 \approx 0$ , our equations simplify to the following:

$$\mathbf{u} = [u_x, u_y]^T. \quad (45)$$

these can be re-written into the following form:

$$\dot{p}_x = v_x, \quad (46)$$

$$\dot{v}_x = -\frac{1}{m} u_x - \frac{1}{m} u_v, \quad (47)$$

$$\dot{r}_{13} = \omega_2, \quad (48)$$

$$\dot{\omega}_1 = -\frac{k_{\omega x}}{J_{11}} \omega_1, \quad (49)$$

$$\dot{\omega}_2 = -\frac{k_{ry}}{J_{22}} r_2 - \frac{k_{\omega y}}{J_{22}} \omega_2 + \frac{b_1 k_{ry}}{J_{22} k_p} e_{px} + \frac{b_2 k_{ry}}{J_{22} k_v} e_{vx} + \frac{a k_{ry}}{J_{22}}, \quad (50)$$

$$\dot{\omega}_3 = -\frac{k_{\omega z}}{J_{33}} \omega_3. \quad (51)$$

As per  $\dot{\theta} = \dot{r}_2 = \omega_2$  and

$$\mathbf{u} = \mathbf{K}_x \mathbf{e}_x = \begin{bmatrix} k_p & 0 \\ 0 & k_v \end{bmatrix} \begin{bmatrix} e_{px} \\ e_{vx} \end{bmatrix}, \quad (52)$$

the system reduces to ( $\omega_1 = 0, \omega_3 = 0$ )

$$\dot{p}_x = v_x, \quad (53)$$

$$\dot{v}_x = -\frac{1}{m} u_x - \frac{1}{m} u_v, \quad (54)$$

$$\dot{\theta} = \omega_2, \quad (55)$$

$$\dot{\omega}_2 = -\frac{k_{\omega y}}{J_{22}} \omega_2 - \frac{k_{ry}}{J_{22}} \theta + \frac{b_1 k_{ry}}{J_{22} k_p} u_x + \frac{b_2 k_{ry}}{J_{22} k_v} v_x + \frac{a k_{ry}}{J_{22}}, \quad (56)$$

where  $a, b_1, b_2$  result from Taylor expansion of  $\theta_d$ .

Let us consider a triplet of external disturbances acting on the system as: an additive linear force, an additive torque, and a another one to model the term  $\frac{ak_{ry}}{J_{22}}$ , in the form

$$\Delta_v, \quad \Delta'_\omega, \quad \Delta_a, \quad (57)$$

respectively, norm-bounded (infinity norm) to 1. Our system of equations can thus be rewritten in the following form:

$$\dot{p}_x = v_x, \quad (58)$$

$$\dot{v}_x = -\frac{1}{m}u_x - \frac{1}{m}u_v + \delta_v\Delta_v, \quad (59)$$

$$\dot{\theta} = \omega_2, \quad (60)$$

$$\dot{\omega}_2 = -\frac{k_{\omega y}}{J_{22}}\omega_2 - \frac{k_{ry}}{J_{22}}\theta + \frac{b_1k_{ry}}{J_{22}k_p}u_x + \frac{b_2k_{ry}}{J_{22}k_v}u_v + \delta_{\omega'}\Delta_{\omega'} + \delta_a\Delta_a. \quad (61)$$

Having assumed zero initial conditions, and incorporating  $\delta_{\omega'}\Delta_{\omega'} + \delta_a\Delta_a$  into  $\delta_\omega\Delta_\omega$ , with  $e_{vx} = \dot{e}_{px}$ , it holds that

$$sP_x(s) = V_x(s), \quad (62)$$

$$sV_x(s) = -\frac{1}{m}U_x(s) - \frac{1}{m}U_v(s) + \delta_v\Delta_v(s), \quad (63)$$

$$s\Theta(s) = \Omega_2(s), \quad (64)$$

$$s\Omega_2(s) = -\frac{k_{\omega y}}{J_{22}}\Omega_2(s) - \frac{k_{ry}}{J_{22}}\Theta(s) + \frac{b_1k_{ry}}{k_pJ_{22}}U_x(s) + \frac{b_2k_{ry}}{k_vJ_{22}}U_v(s) + \delta_\omega\Delta_\omega(s). \quad (65)$$

where  $u_v = k_v e_{vx} = k_v \dot{e}_{px} = k \dot{u}_x = k u_x$  with  $k = \frac{k_v}{k_p}$ .

The above systems can be reduced as follows:

$$V_x(s) = -\frac{ks+1}{sm}U_x(s) + \frac{\delta_v}{s}\Delta_v(s), \quad (66)$$

$$s\Omega_2(s) = -\frac{1}{J_{22}}\left(\frac{k_{ry}}{s} + k_{\omega y}\right)\Omega_2(s) + \frac{1}{k_pJ_{22}}(sb_2k_{ry} + b_1k_{ry})U_x(s) + \delta_\omega\Delta_\omega(s) \quad (67)$$

and

$$V_x(s) = -\frac{ks+1}{sm}U_x(s) + \frac{\delta_v}{s}\Delta_v(s), \quad (68)$$

$$\Omega_2(s) = \frac{s^2b_2k_{ry} + sb_1k_{ry}}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}}U_x(s) + \frac{sk_pJ_{22}\delta_\omega}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}}\Delta_\omega(s). \quad (69)$$

what can be presented as

$$\begin{bmatrix} V_x(s) \\ \Omega_2(s) \end{bmatrix} = \begin{bmatrix} -\frac{ks+1}{sm} & \frac{\delta_v}{s} & 0 \\ \frac{s^2b_2k_{ry} + sb_1k_{ry}}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}} & 0 & \frac{sk_pJ_{22}\delta_\omega}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}} \end{bmatrix} \begin{bmatrix} U_x(s) \\ \Delta_v(s) \\ \Delta_\omega(s) \end{bmatrix}. \quad (70)$$

Taking the connection between  $U_x(s)$  and  $P_x(s)$  in the form

$$L_1(s) = -k_P \frac{ks+1}{sm} \quad (71)$$

$$T_1(s) = \frac{L_1(s)}{1+L_1(s)} = -\frac{k_P \frac{ks+1}{sm}}{1 - k_P \frac{ks+1}{sm}} = \quad (72)$$

$$= -\frac{k_P(ks+1)}{ms - k_P(ks+1)} = \frac{k_v s + k_p}{(k_v - m)s + k_p}, \quad (73)$$

$$S_1(s) = \frac{1}{1+L_1(s)} = -\frac{sm}{(k_v - m)s + k_p}, \quad (74)$$

and also between  $U_v(s) = sU_x(s)$  and  $\Omega_2(s)$  in the form

$$L_2(s) = k_v \frac{sb_2k_{ry} + b_1k_{ry}}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}}, \quad (75)$$

$$T_2(s) = \frac{L_2(s)}{1+L_2(s)} = \frac{k_v \frac{sb_2k_{ry} + b_1k_{ry}}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}}}{1 + k_v \frac{sb_2k_{ry} + b_1k_{ry}}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}}} = \quad (76)$$

$$= \frac{sk_v b_2 k_{ry} + k_v b_1 k_{ry}}{s^2 k_p J_{22} + s k_p k_{\omega y} + k_p k_{ry} + s k_v b_2 k_{ry} + k_v b_1 k_{ry}} \quad (77)$$

$$= \frac{sk_v b_2 k_{ry} + k_v b_1 k_{ry}}{s^2 k_p J_{22} + s(k_p k_{\omega y} + k_v b_2 k_{ry}) + k_{ry}(k_p + k_v b_1)} \quad (78)$$

$$S_2(s) = \frac{1}{1 + k_v \frac{sb_2k_{ry} + b_1k_{ry}}{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}}} = \quad (79)$$

$$= \frac{s^2k_pJ_{22} + sk_pk_{\omega y} + k_pk_{ry}}{s^2k_pJ_{22} + s(k_p k_{\omega y} + k_v b_2 k_{ry}) + k_{ry}(k_p + k_v b_1)} \quad (80)$$

At this point, naturally the following constraints appear, related to Routh stability criterion:

$$k_v - m > 0, \quad (81)$$

$$k_p > 0, \quad (82)$$

$$k_p + k_v b_1 > 0, \quad (83)$$

$$k_p k_{\omega y} + k_v b_2 k_{ry} > 0, \quad (84)$$

as

$$k_v > m, \quad (85)$$

$$k_p > 0, \quad (86)$$

giving natural bounds on controller gains. Please note that the transfer functions are of second-order, and as per Nyquist stability criterion, the closed-loop system has a simple stability conditions.

In the potential analysis, and following [26], a MULTIPLICATIVE uncertainty  $\Delta(s)$  acting at a plant  $P(s)$  output can be included, the transfer function between the output of the uncertainty and its input equals  $-T_1(s)$  or  $-T_2(s)$ . By the small gain theorem, these sub-systems remain stable for

$$\|\Delta(j\omega)\|_\infty \|T(j\omega)\|_\infty < 1 \quad (87)$$

what corresponds to all the plants in the form  $P(s)(1 + \Delta(s))$  for  $\|\Delta(j\omega)\|_\infty < r$  where  $\|T(j\omega)\|_\infty < \frac{1}{r}$ .

Considering an ADDITIVE uncertainty  $\Delta(s)$  acting on the plant  $P(s)$ , the transfer function between the output of the uncertainty and its input equals  $C(s)S_1(s)$  or  $C(s)S_2(s)$ , where the plant is given by  $P(s) + \Delta(s)$  and  $C(s)$  is the transfer function of the controller. By the small gain theorem, these sub-systems remain stable for

$$\|\Delta(j\omega)\|_\infty \|C(j\omega)S(j\omega)\|_\infty < 1. \quad (88)$$

Abiding the constraints, the closed-loop systems remain stable whenever in the model ( $m = 1.8$ ,  $k_{ry} = 7$ ,  $k_{\omega y} = 0.7$ ,  $J_{22} = 11.4 \cdot 11^{-6}$ ) it holds that  $k_p > 0$  and  $k_v > m$ .

It can be seen from Figures 7 and 8 that  $\|T_1\|_\infty$  and  $\|T_2\|_\infty$  are always norm-bounded,  $\|T_2\|_\infty$  is always below 1, while  $\|T_1\|_\infty$  is below 1 for low frequencies. As such, the linear controller is robust to the aforementioned external disturbances, while the attitude controller is robust to these disturbances only when these disturbances have a low frequency.

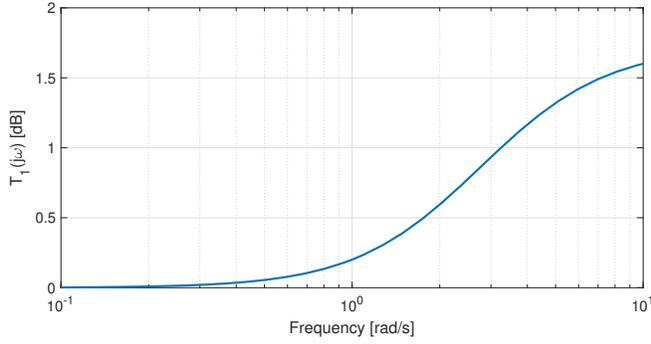


Fig. 7. Complementary sensitivity function  $T_1(j\omega)$  for  $\overline{f_{xy}} = 4$ ,  $k_p = 25$ ,  $k_v = 10$

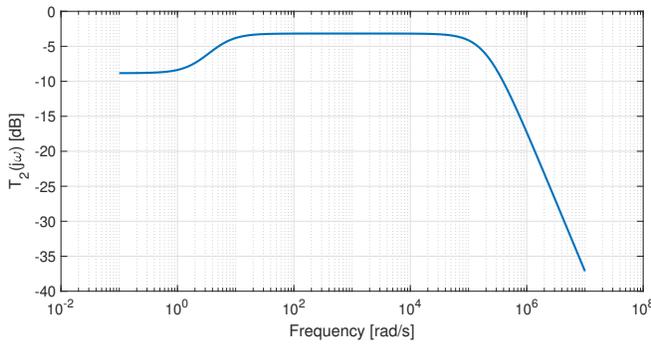


Fig. 8. Complementary sensitivity function  $T_2(j\omega)$  for  $\overline{f_{xy}} = 4$ ,  $k_p = 25$ ,  $k_v = 10$

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